



Identification of fire suppression mechanisms by water mist: experimental and numerical analysis

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Extinction mechanisms

- Gas phase cooling;
- Oxygen displacement and fuel vapor dilution;
- Fuel surface cooling and wetting;

- Radiative transfer attenuation;
- Kinetic effects.



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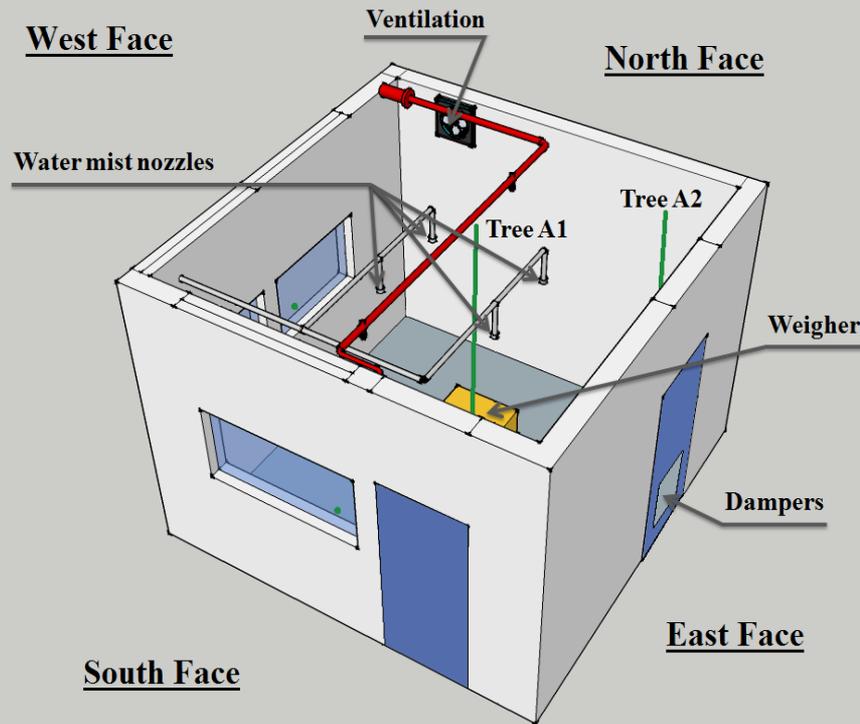
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Experimental setup



Metrology:

- 18 thermocouples
- Gas analyser for O₂, CO₂ and CO
- Load cell
- Video camera

Water mist characteristics:

- Pressure = 10 bars
- Flow rate = 6.3 l/min/nozzle
- Injection angle = 130°
- D₃₂ = 112 μm

- Pool diameters : 25 and 35 cm
- HRR up to 75 kW





Extinction mechanisms observation

Test 18

$t_{app} = 5 \text{ min}$



$t_0 + 10 \text{ s}$



$t_0 + 1 \text{ min}$



$t_0 + 2 \text{ min}$



$t_0 + 3 \text{ min}$



$t_{app} - 1 \text{ s}$



$t_{app} + 2 \text{ s}$



$t_{app} + 5 \text{ s}$



$t_{app} + 8 \text{ s}$



$t_{app} + 10 \text{ s}$

- Late application: strong evaporation;
- Dominating mechanism: flame cooling and inerting effects;
- Quick extinction.



Extinction mechanisms observation

Test 14

$t_{app} = 1 \text{ min}$



$t_0 + 10 \text{ s}$



$t_{app} - 1 \text{ s}$



$t_{app} + 2 \text{ s}$



$t_{app} + 5 \text{ s}$



$t_{app} + 10 \text{ s}$



$t_{app} + 20 \text{ s}$



$t_{app} + 30 \text{ s}$



$t_{app} + 60 \text{ s}$



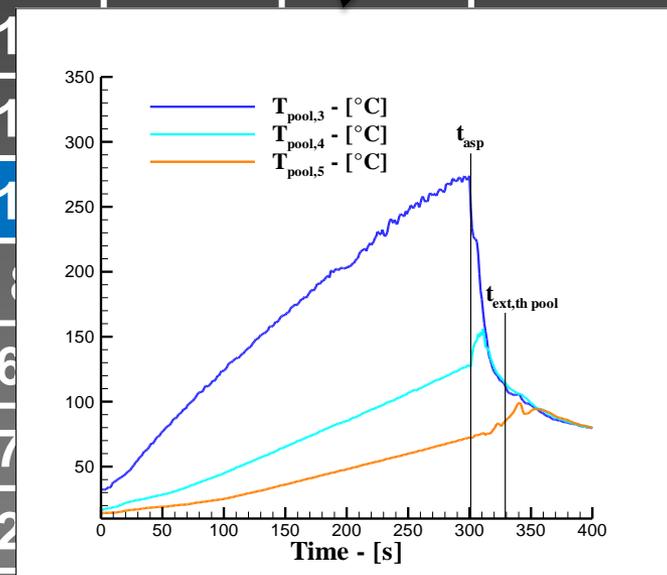
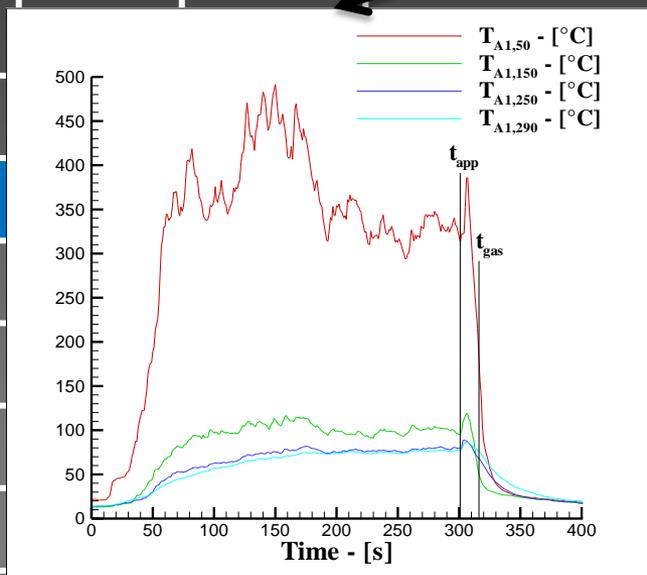
$t_{app} + 65 \text{ s}$

- Early application: weak evaporation;
- Dominating mechanism: fuel cooling;
- Longer extinction.



Experimental results: fuel oil

N°	D _{pool} [cm]	T ₀ [°C]	t _{app} [s]	Q _{app} [kW]	T _{A2,290,app} [°C]	Δt _{vid} [s]	Δt _{gas} [s]	Δt _{pool} [s]	Ext. Mech.
13	35	12	38	8	21	40	35	35	F.C
14	35	11	64	15	32	65	61	65	F.C



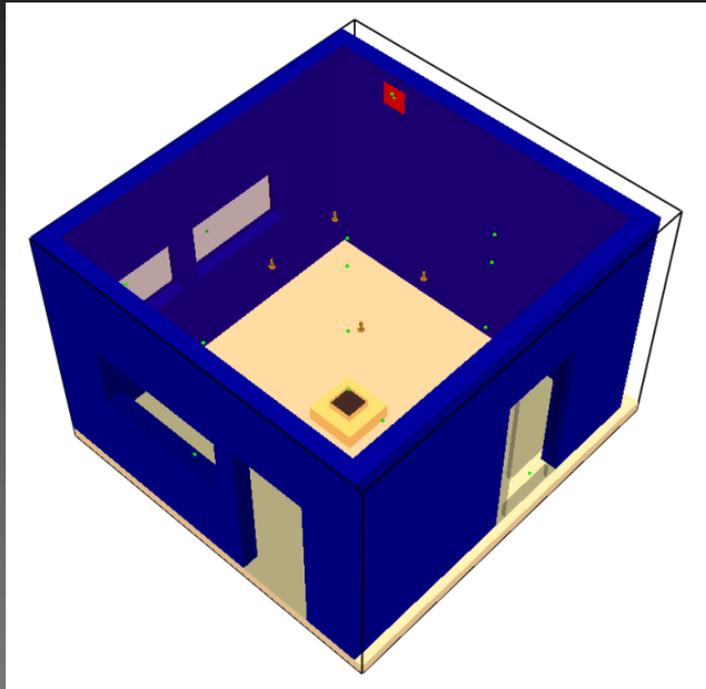


Experimental results: fuel oil

N°	D _{pool} [cm]	T ₀ [°C]	t _{app} [s]	Q _{app} [kW]	T _{A2,290,app} [°C]	Δt _{vid} [s]	Δt _{gas} [s]	Δt _{pool} [s]	Ext. Mech.
<p>A hotter environment leads to « gas cooling » extinction and a cooler to « fuel cooling » extinction</p> <p>For « fuel cooling » tests, Δt_{vid} ≈ Δt_{pool}</p> <p>For « gas cooling » tests, Δt_{vid} << Δt_{pool}</p>					21	40	35	35	F.C
					32	65	61	65	F.C
					52	10	8	24	G.C
					61	16	17	61	G.C
					71	10	15	28	G.C
					52	8	10	30	G.C
					42	65	58	78	F.C
					51	70	57	78	F.C
					62	26	26	56	G.C



Numerical model

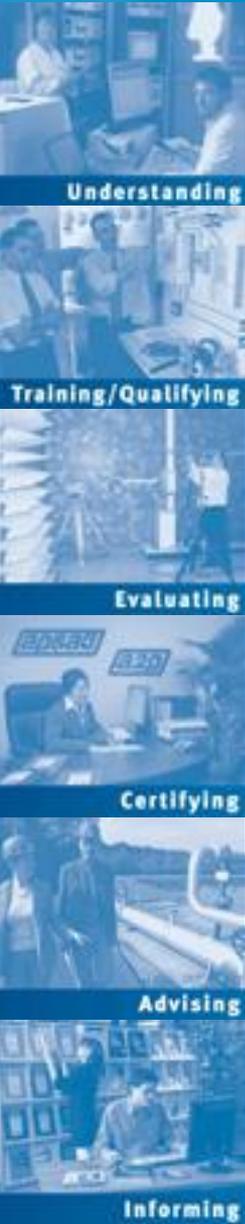


Main parameters:

- Cell size: 5 cm x 5 cm x 5 cm;
- HRR increase defined as a ramp following the actual measured curve until stationary regime;
- After mist activation, HRR guided toward a reduction through extinction model.

$$\text{Extinction model: } \dot{m}''_{pyro}(t) = \dot{m}''_{pyro,0} e^{-\int k(t)dt}$$

with $k(t) = a m''_w(t)$





Mass and energy balances

Mass balances

For « flame + gas »

$$\dot{m}_g(t) = \dot{m}_{pyro}(t) + \dot{m}_{adv,g}(t) + \dot{m}_{evap}(t)$$

For « droplets »

$$\dot{m}_p(t) = \dot{m}_{inj}(t) + \dot{m}_{adv,p}(t) - \dot{m}_{evap}(t)$$

Energy balances

For « flame + gas »

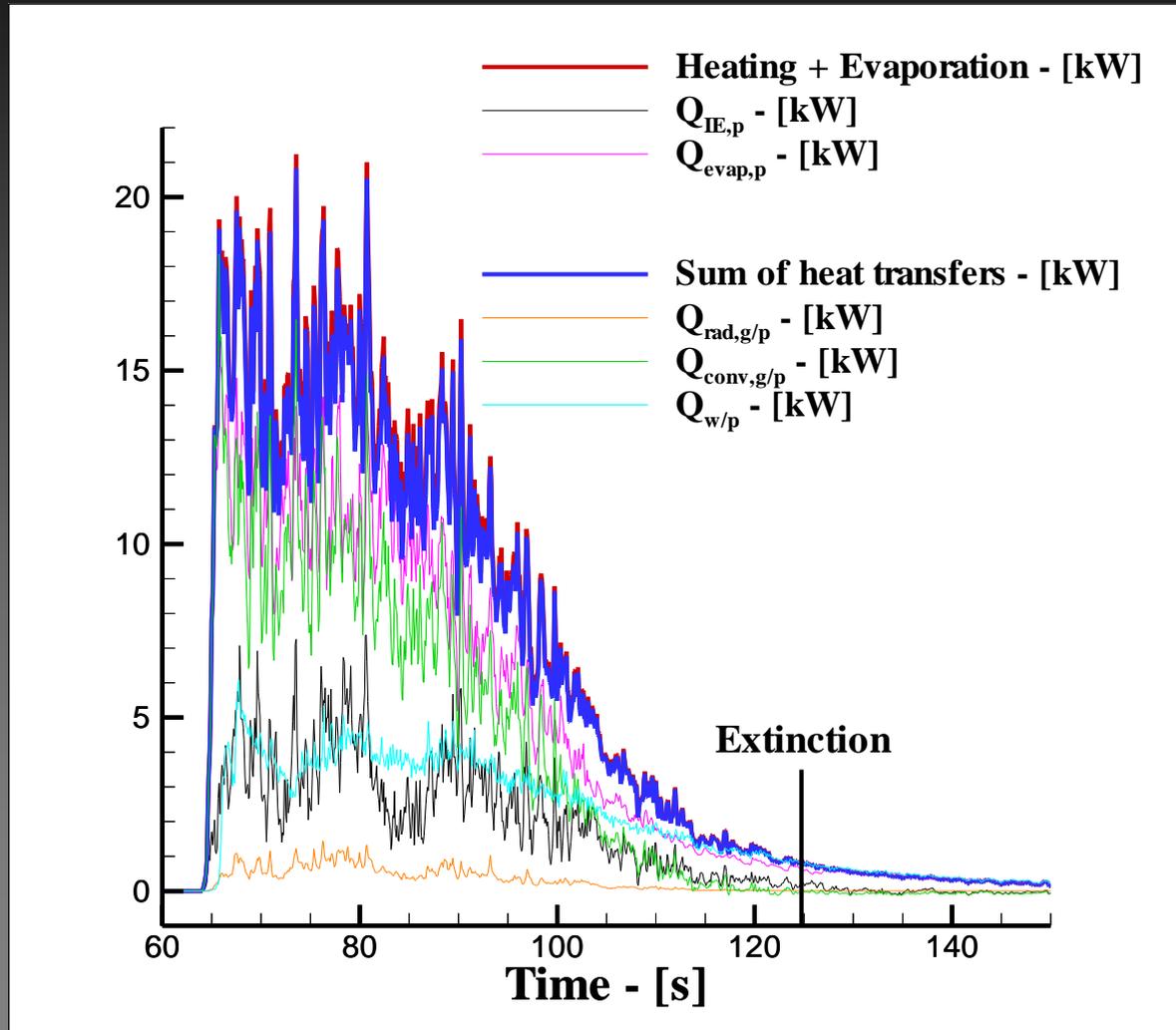
$$\begin{aligned} \dot{Q}(t) = & \dot{Q}_{conv,g/w}(t) + \dot{Q}_{rad,g/w}(t) + \dot{Q}_{adv,g}(t) \\ & + \dot{Q}_{conv,p/g}(t) + \dot{Q}_{rad,p/g}(t) + \dot{Q}_{IE,g}(t) \end{aligned}$$

For « droplets »

$$\begin{aligned} & \dot{Q}_{evap}(t) + \dot{Q}_{IE,p}(t) \\ = & \dot{Q}_{conv,g/p}(t) + \dot{Q}_{rad,g/p}(t) + \dot{Q}_{w/p}(t) \end{aligned}$$



Energy balances results





Fire extinction model

HRR reduction should be linked to fuel temperature during water application

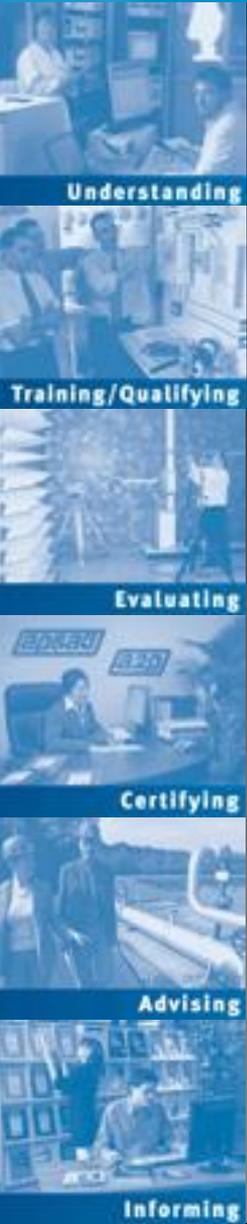
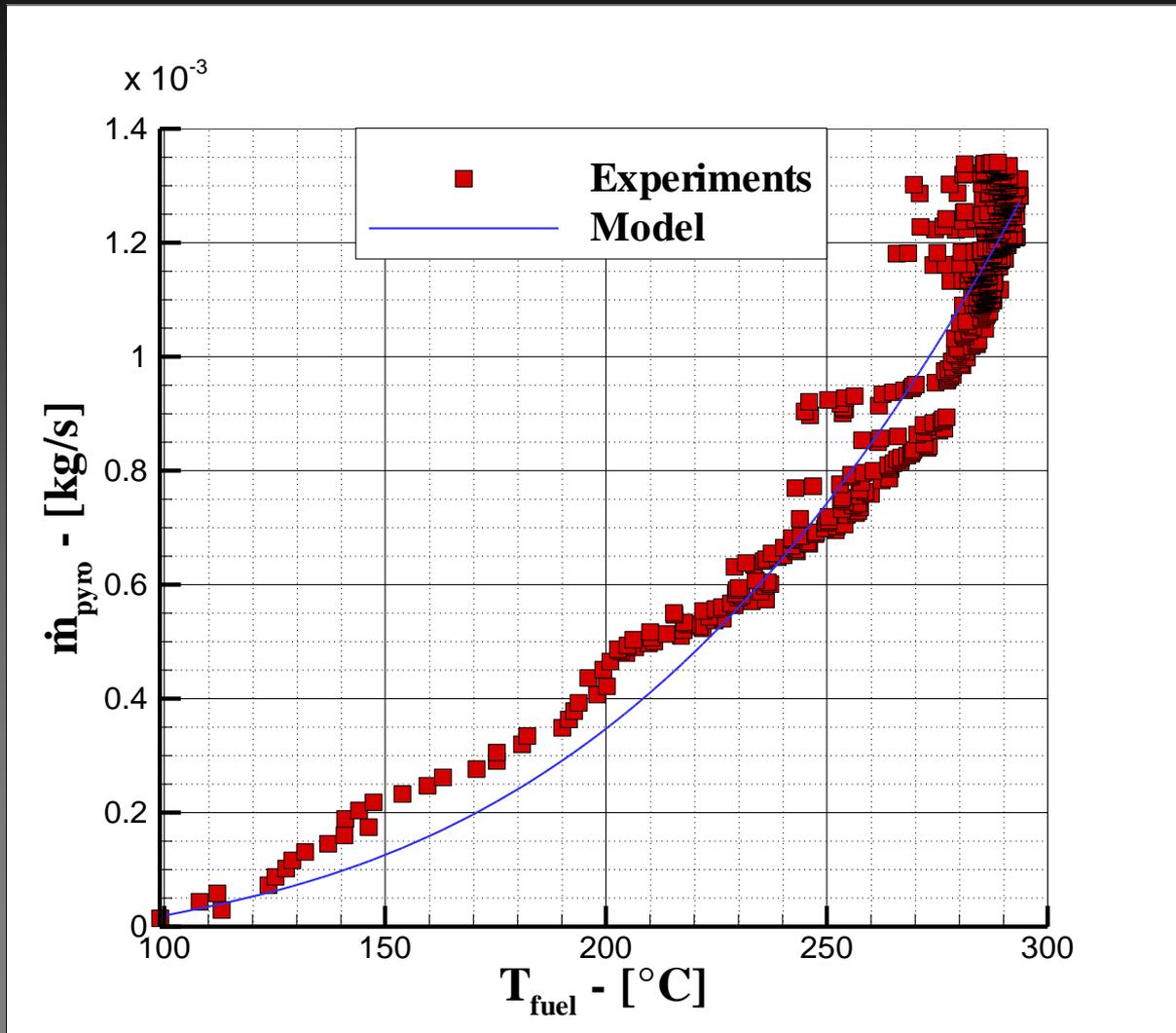
$$\text{Model: } \begin{cases} \dot{m}''_{pyro}(t) = A \sqrt{T_{fuel}(t) - T_{ign}} \exp\left(-\frac{B}{T_{fuel}(t)}\right) & \text{if } T_{fuel} \geq T_{ign} \\ \dot{m}''_{pyro}(t) = 0 & \text{if } T_{fuel} < T_{ign} \end{cases}$$

A and B are empirical coefficients which can be easily determined, even without experimental tests.

$$\text{Based on the Arrhenius law: } \dot{m}''_{pyro}(t) = A' \exp\left(-\frac{B'}{T_{fuel}(t)}\right)$$

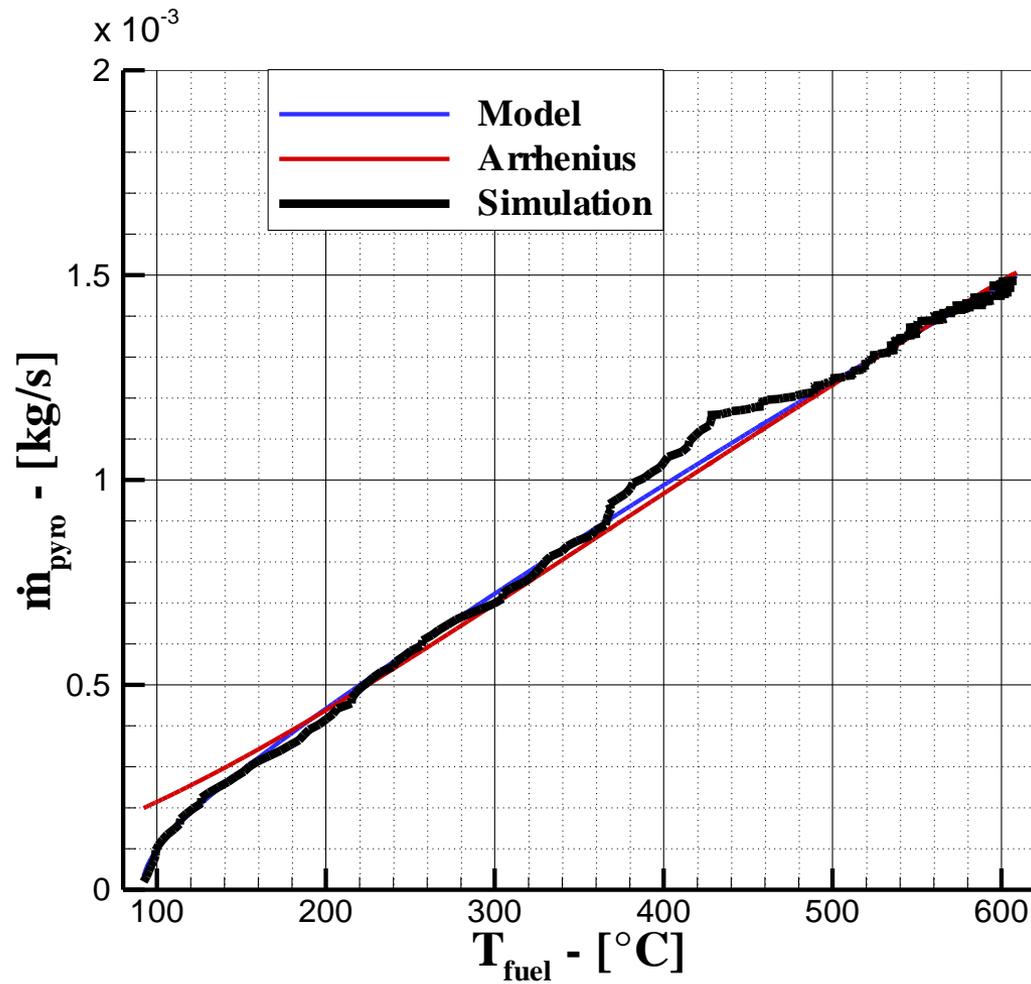


Validation without water mist



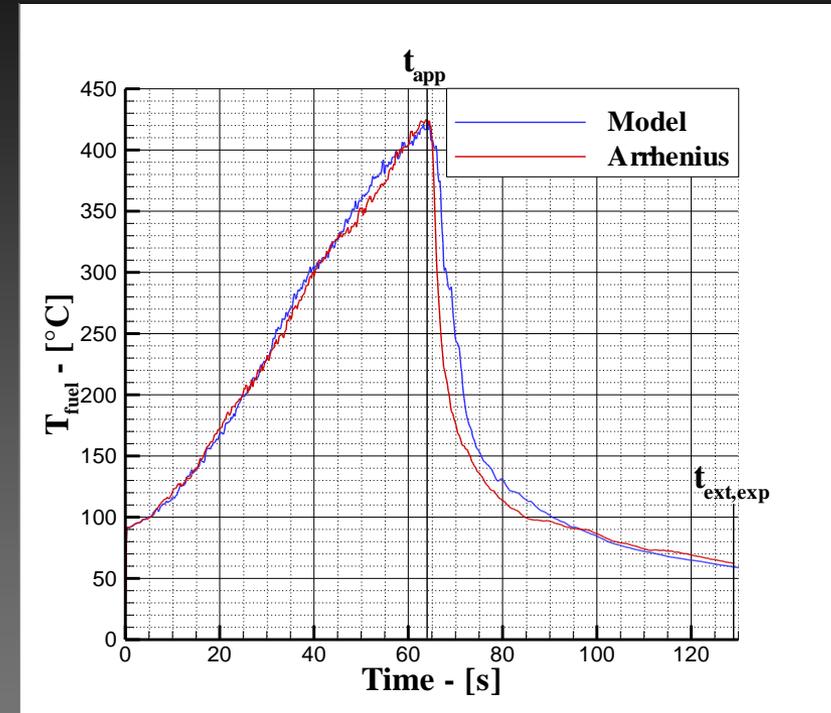
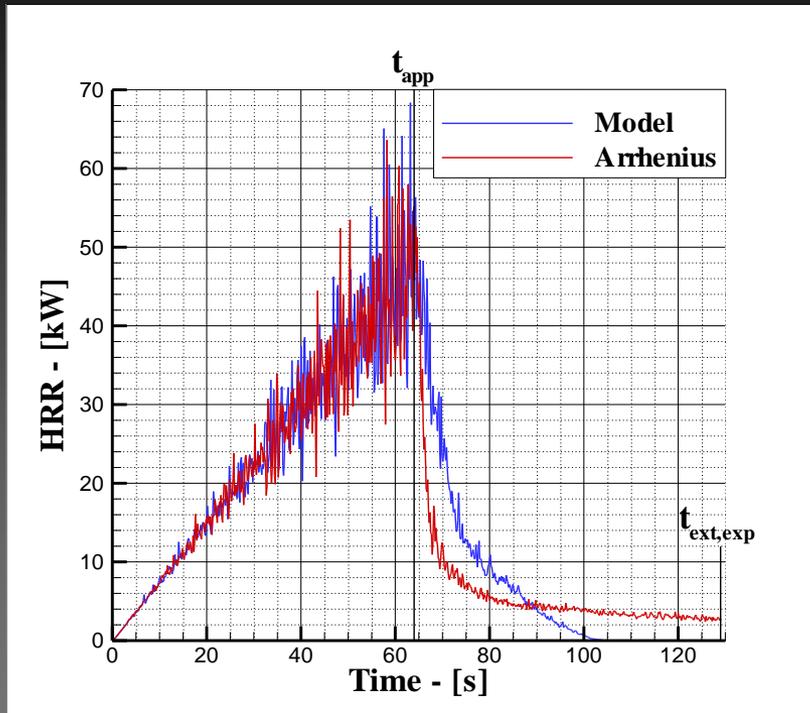


With simulation only





Results on extinction time



- Identical results for T_{fuel} but extinction after 38 s with model.
- $t_{ext,num}$ too short ($t_{ext,exp} = 65$ s), corresponding to a gap of 42 %.
- First results with a full predictive model.



Results on extinction time

N°	13	14	20	21	31	32	33	34	35	36	37
$t_{\text{ext,exp}}$ (s)	40	65	65	70	31	30	69	99	106	105	129
$t_{\text{ext,num}}$ (s)	22	38	56	66	9	21	43	58	77	100	134
Gap (s)	18	27	9	4	22	9	26	41	29	5	5
Gap (%)	45	42	14	6	71	30	38	41	27	5	4

- HRR reduction is always captured;
- Gap between 4 and 41 s between experimental and model results;
- Model does not predict “wrong” extinction.



Conclusions

1. Late application => Developed fire and hot environment => Strong evaporation => Quick extinction by gas cooling and inerting effects;
2. Early application => “Cool” environment => Weak evaporation => Longer extinction by fuel cooling for fuel oil fires;
3. For the latter case, a new extinction model has been integrated to FDS;
4. The average gap between experimental and model results is 18 s (or 29 %). The model did not predict “wrong” extinction.



Current work

1. Improve model results through a modification of particles / fuel exchanges modeling;
2. Determine FDS capability to determine extinction by flame cooling and inerting effects in FDS 6.



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Energy balances results

N°	a [m ² /kg/s]	$\frac{\dot{Q}_{evap}}{\dot{Q}_p}$	$\frac{\dot{Q}_{IE,p}}{\dot{Q}_p}$	$\frac{\dot{Q}_{conv,g/p}}{\dot{Q}_p}$	$\frac{\dot{Q}_{rad,g/p}}{\dot{Q}_p}$	$\frac{\dot{Q}_{w/p}}{\dot{Q}_p}$	Ext. Mech.
13	0.3	68 %	32 %	58 %	4 %	38 %	F.C
14	0.15	74 %					F.C
16	50	84 %					G.C
17	20	86 %					G.C
18	30	87 %					G.C
19	90	83 %					G.C
20	0.6	75 %					F.C
21	0.2	76 %					F.C
22	3.3	81 %	19 %	71 %	4 %	25 %	G.C

Strong evaporation for « gas cooling » cases. In any case, evaporation is stronger than variation of internal energy.

‘a’ between 0 and 1 m²/kg/s for (suited) « fuel cooling » cases.



Energy balances results

N°	a [m ² /kg/s]	$\frac{\dot{Q}_{evap}}{\dot{Q}_p}$	$\frac{\dot{Q}_{IE,p}}{\dot{Q}_p}$	$\frac{\dot{Q}_{conv,g/p}}{\dot{Q}_p}$	$\frac{\dot{Q}_{rad,g/p}}{\dot{Q}_p}$	$\frac{\dot{Q}_{w/p}}{\dot{Q}_p}$	Ext. Mech.
13	0.3	68 %	32 %	58 %	4 %	38 %	F.C
14	0.15	74 %	26 %	62 %	4 %	34 %	F.C
16	50	84 %	16 %	74 %	3 %	23 %	G.C
17	20	86 %	14 %	79 %	4 %	17 %	G.C
18	30	87 %	13 %	73 %	3 %	24 %	G.C
19	90	83 %	17 %	65 %	2 %	33 %	G.C
20	0.6	75 %	25 %	71 %	4 %	24 %	F.C
21	0.2	76 %	24 %	67 %	5 %	28 %	F.C
22	3.3	81 %	19 %	71 %	4 %	25 %	G.C



Energy balances results

N°	a [m ² /kg/s]	$\frac{\dot{Q}_{evap}}{\dot{Q}_p}$	$\frac{\dot{Q}_{IE,p}}{\dot{Q}_p}$	$\frac{\dot{Q}_{conv,g/p}}{\dot{Q}_p}$	$\frac{\dot{Q}_{rad,g/p}}{\dot{Q}_p}$	$\frac{\dot{Q}_{w/p}}{\dot{Q}_p}$	Ext. Mech.
31	1	65 %	35 %	57 %	4 %	39 %	F.C
32	1	71 %	29 %	61 %	4 %	35 %	F.C
33	0.24	75 %	25 %	68 %	5 %	27 %	F.C
34	0.26	77 %	23 %	74 %	6 %	20 %	F.C
35	0.42	77 %	23 %	78 %	6 %	15 %	F.C
36	0.74	79 %	21 %	81 %	7 %	12 %	F.C
37	0.24	79 %	21 %	81 %	7 %	13 %	F.C

- Evaporation part < 80 %;
- 'a' between 0 and 1 m²/kg/s;
- Convection > wall / particles exchanges > radiation.